### SFL

# **ENSC 283 Quiz #2**

March 24, 2009

Name: ...... Student ID:.....

Time: 45 minutes or less. Develop answers on available place. The quiz has 5% (bonus) of the total mark. Closed books & closed notes.

## **Problem 1 (40%)**:

The velocity field near a stagnation point may be written in the form:

$$u = \frac{U_0 x}{L}$$
  $v = \frac{U_0 y}{L}$   $U_0$  and  $L$  are constants

- a) Show that the acceleration vector is purely radial.
- b) For the particular case L=1.5 m, if the acceleration at (x,y)=(1m, 1m) is 25 m/s<sup>2</sup>, what is the value of  $U_0$ ?

Acceleration:

$$\frac{du}{dt} = \frac{\partial u}{\partial t} + u \frac{\partial u}{\partial x} + v \frac{\partial u}{\partial y} + w \frac{\partial u}{\partial z}$$
$$\frac{dv}{dt} = \frac{\partial v}{\partial t} + u \frac{\partial v}{\partial x} + v \frac{\partial v}{\partial y} + w \frac{\partial v}{\partial z}$$

#### **Solution:**

(a) For two-dimensional steady flow, the acceleration components are

$$\frac{du}{dt} = u\frac{\partial u}{\partial x} + v\frac{\partial u}{\partial v} = \frac{U_0^2}{L^2}x$$

$$\frac{dv}{dt} = u\frac{\partial v}{\partial x} + v\frac{\partial v}{\partial v} = \frac{U_0^2}{L^2}y$$

Therefore the resultant acceleration would be

$$a = \frac{U_0^2}{I_0^2}(xi + yj) = \frac{U_0^2}{I_0^2}r$$

which purely radial. Its magnitude would be

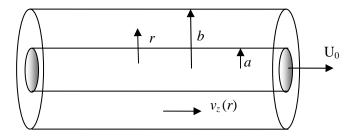
$$a = \frac{U_0^2}{I^2} \sqrt{x^2 + y^2}$$

(b) For a given resultant acceleration of 25  $m/s^2$ , at (x, y) = (1m, 1m), we obtain

$$25 = \frac{U_0^2}{1.5^2} \sqrt{1^2 + 1^2} \rightarrow U_0^2 = \frac{25 \times 1.5^2}{\sqrt{2}} \text{ or } U_0 = 6.3 \text{ m/s}$$

## Problem 2 (60%):

The viscous oil in figure below is set into steady motion by a concentric inner cylinder moving axially at velocity  $U_0$  inside a fixed outer cylinder. Assuming constant pressure and density and a purely axial fluid motion, solve the axial momentum equation (given below) for the velocity distribution  $v_z(r)$ . What are the proper boundary conditions?



The continuity equation in cylindrical coordinates:

$$\frac{1}{r}\frac{\partial}{\partial r}(rv_r) + \frac{1}{r}\frac{\partial}{\partial \theta}(v_\theta) + \frac{\partial}{\partial z}(v_z) = 0$$

The z-momentum equation in cylindrical coordinates:

$$\rho\left(\frac{\partial v_z}{\partial t} + v_r \frac{\partial v_z}{\partial r} + \frac{v_\theta}{r} \frac{\partial v_z}{\partial \theta} + v_z \frac{\partial v_z}{\partial z}\right) = -\frac{\partial p}{\partial z} + \mu \left[\frac{1}{r} \frac{\partial}{\partial r} \left(r \frac{\partial v_z}{\partial r}\right) + \frac{1}{r^2} \frac{\partial^2 v_z}{\partial \theta^2} + \frac{\partial^2 v_z}{\partial z^2}\right] + \rho g_z$$

#### Solution:

assumptions:

- Laminar flow
- Steady state
- Gravity in the z direction is negligible
- Constant pressure
- Constant properties
- Purely axial motion

Using these assumptions,  $v_z(r)$  is only a function of r and momentum equation simplifies to

$$\frac{\mu}{r}\frac{d}{dr}\left(r\frac{dv_z}{dr}\right) = 0$$

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subjected to the following boundary conditions

$$v_z(a) = U_0; \ v_z(0) = 0$$

The solution for the differential equation is

$$v_z = C_1 \ln(r) + C_2$$

Applying the boundary conditions the final solution is

$$v_z = U_0 \frac{\ln\left(\frac{r}{b}\right)}{\ln\left(\frac{a}{b}\right)}$$