

Solution: (a) The resultant force F , may be found by simply applying the hydrostatic relation

$$F = \gamma h_{CG} A = (9790 \text{ N/m}^3)(3 + 1.5 \text{ m})(5 \text{ m} \times 2 \text{ m}) = 440,550 \text{ N} = \mathbf{441 \text{ kN}} \quad \text{Ans. (a)}$$

(b) The horizontal force acts as though BC were vertical, thus h_{CG} is halfway down from C and acts on the projected area of BC .

$$F_H = (9790)(4.5)(3 \times 2) = 264,330 \text{ N} = \mathbf{264 \text{ kN}} \quad \text{Ans. (b)}$$

The vertical force is equal to the weight of fluid above BC ,

$$F_V = (9790)[(3)(4) + (1/2)(4)(3)](2) = 352,440 = \mathbf{352 \text{ kN}} \quad \text{Ans. (b)}$$

The resultant is the same as part (a): $F = [(264)^2 + (352)^2]^{1/2} = \mathbf{441 \text{ kN}}$.

2.58 In Fig. P2.58, weightless cover gate AB closes a circular opening 80 cm in diameter when weighed down by the 200-kg mass shown. What water level h will dislodge the gate?

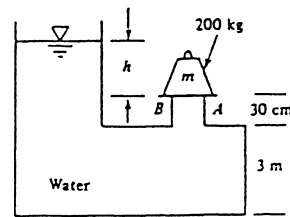


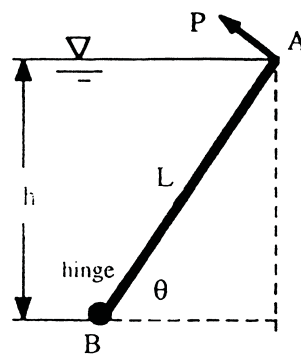
Fig. P2.58

Solution: The centroidal depth is exactly equal to h and force F will be upward on the gate. Dislodging occurs when F equals the weight:

$$F = \gamma h_{CG} A_{\text{gate}} = (9790 \text{ N/m}^3) h \frac{\pi}{4} (0.8 \text{ m})^2 = W = (200)(9.81) \text{ N}$$

$$\text{Solve for } h = \mathbf{0.40 \text{ m}} \quad \text{Ans.}$$

2.59 Gate AB has length L , width b into the paper, is hinged at B , and has negligible weight. The liquid level h remains at the top of the gate for any angle θ . Find an analytic expression for the force P , perpendicular to AB , required to keep the gate in equilibrium.



Solution: The centroid of the gate remains at distance $L/2$ from A and depth $h/2$ below

the surface. For any θ , then, the hydrostatic force is $F = \gamma(h/2)Lb$. The moment of inertia of the gate is $(1/12)bL^3$, hence $y_{CP} = -(1/12)bL^3 \sin\theta / [(h/2)Lb]$, and the center of pressure is $(L/2 - y_{CP})$ from point B. Summing moments about hinge B yields

$$PL = F(L/2 - y_{CP}), \quad \text{or:} \quad \mathbf{P = (\gamma hb/4)(L - L^2 \sin \theta/3h)} \quad \text{Ans.}$$

P2.60 In 1960, Auguste and Jacques Picard's self-propelled bathyscaphe *Trieste* set a record by descending to a depth of 35,800 feet in the Pacific Ocean, near Guam. The passenger sphere was 7 ft in diameter, 6 inches thick, and had a window 16 inches in diameter. (a) Estimate the hydrostatic force on the window at that depth. (b) If the window is vertical, how far below its center is the center of pressure?

Solution: At the surface, the density of seawater is about 1025 kg/m^3 (1.99 slug/ft^3). Atmospheric pressure is about 2116 lbf/ft^2 . We could use these values, or estimate from Eq. (1.19) that the density at depth would be about 4.6% more, or 2.08 slug/ft^3 . We could average these two to 2.035 slug/ft^3 . The pressure at that depth would thus be approximately

$$p = p_a + \rho_{avg} g h = 2116 + (2.035 \frac{\text{slug}}{\text{ft}^3})(32.2 \frac{\text{ft}}{\text{s}^2})(35800 \text{ ft}) \approx 2,350,000 \frac{\text{lbf}}{\text{ft}^2}$$

(a) This pressure, times the area of the 16-inch window, gives the desired force.

$$F_{\text{window}} = p_{CG} A = (2350000 \frac{\text{lbf}}{\text{ft}^2}) [\frac{\pi}{4} (\frac{16}{12} \text{ ft})^2] = \mathbf{3,280,000 \text{ lbf}} \quad \text{Ans.(a)}$$

Quite a lot of force, but the bathyscaphe was well designed.

(b) The distance down to the center of pressure on the window follows from Eq. (2.27):

$$y_{CP} = -\gamma \sin \theta \frac{I_{xx}}{F} = -[2.035 * 32.2 \frac{\text{lbf}}{\text{ft}^3}] \sin(90^\circ) \frac{(\pi/4)(8/12 \text{ ft})^4}{3280000 \text{ lbf}} = \mathbf{-3.2E - 6 \text{ ft.}} \quad \text{Ans.(b)}$$

The center of pressure at this depth is only 38 micro inches below the center of the window.
