

(b) At the wall,  $u$  must be approximately linear with  $y$ , if  $\tau_w \geq 0$ :

Near wall:  $u \approx y f(x)$ , or  $\frac{\partial u}{\partial x} = y \frac{df}{dx}$ , where  $\frac{df}{dx} < 0$ . Then  $\frac{\partial v}{\partial y} = -\frac{\partial u}{\partial x} = \left(\frac{df}{dx}\right) y$

Thus, near the wall,  $v \approx \left(\frac{df}{dx}\right) \int_0^y y dy \approx \left(\frac{df}{dx}\right) \frac{y^2}{2}$  **Parabolic** Ans. (b)

(c) At  $y = \delta$ ,  $u \rightarrow U$ , then  $\partial u / \partial x \approx 0$  there and thus  $\partial v / \partial y \approx 0$ , or  $v = v_{\max}$ . Ans. (c)

**4.25** An incompressible flow in polar coordinates is given by

$$v_r = K \cos \theta \left(1 - \frac{b}{r^2}\right)$$

$$v_\theta = -K \sin \theta \left(1 + \frac{b}{r^2}\right)$$

Does this field satisfy continuity? For consistency, what should the dimensions of constants  $K$  and  $b$  be? Sketch the surface where  $v_r = 0$  and interpret.

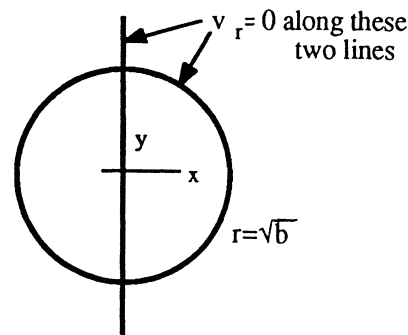


Fig. P4.25

**Solution:** Substitute into plane polar coordinate continuity:

$$\frac{1}{r} \frac{\partial}{\partial r} (r v_r) + \frac{1}{r} \frac{\partial v_\theta}{\partial \theta} = 0 \stackrel{?}{=} \frac{1}{r} \frac{\partial}{\partial r} \left[ K \cos \theta \left( r - \frac{b}{r} \right) \right] + \frac{1}{r} \frac{\partial}{\partial \theta} \left[ -K \sin \theta \left( 1 + \frac{b}{r^2} \right) \right] = 0 \text{ Satisfied}$$

The dimensions of  $K$  must be velocity,  $\{K\} = \{L/T\}$ , and  $b$  must be area,  $\{b\} = \{L^2\}$ . The surfaces where  $v_r = 0$  are the  $y$ -axis and the circle  $r = \sqrt{b}$ , as shown above. The pattern represents inviscid flow of a uniform stream past a circular cylinder (Chap. 8).

**4.26** Curvilinear, or streamline, coordinates are defined in Fig. P4.26, where  $n$  is normal to the streamline in the plane of the radius of curvature  $R$ . Show that Euler's frictionless momentum equation (4.36) in streamline coordinates becomes

$$-V \frac{\partial \theta}{\partial t} - \frac{V^2}{R} = -\frac{1}{\rho} \frac{\partial p}{\partial n} + g_n \quad (2)$$

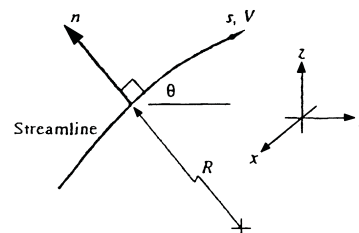


Fig. P4.26

Further show that the integral of Eq. (1) with respect to  $s$  is none other than our old friend Bernoulli's equation (3.76).

**Solution:** This is a laborious derivation, really, **the problem is only meant to acquaint the student with streamline coordinates.** The second part is not too hard, though. Multiply the streamwise momentum equation by  $ds$  and integrate:

$$\frac{\partial \mathcal{V}}{\partial t} ds + V dV = -\frac{dp}{\rho_2} + g_s ds = -\frac{dp}{\rho} - g \sin \theta ds = -\frac{dp}{\rho} - g dz$$

Integrate from 1 to 2:  $\int_1^2 \frac{\partial \mathcal{V}}{\partial t} ds + \frac{V_2^2 - V_1^2}{2} + \int_1^2 \frac{dp}{\rho} + g(z_2 - z_1) = 0$  (Bernoulli) *Ans.*

**4.27** A frictionless, incompressible steady-flow field is given by

$$\mathbf{V} = 2xy\mathbf{i} - y^2\mathbf{j}$$

in arbitrary units. Let the density be  $\rho_0 = \text{constant}$  and neglect gravity. Find an expression for the pressure gradient in the  $x$  direction.

**Solution:** For this (gravity-free) velocity, the momentum equation is

$$\rho \left( u \frac{\partial \mathcal{V}}{\partial x} + v \frac{\partial \mathcal{V}}{\partial y} \right) = -\nabla p, \quad \text{or: } \rho_0 [(2xy)(2y\mathbf{i}) + (-y^2)(2x\mathbf{i} - 2y\mathbf{j})] = -\nabla p$$

Solve for  $\nabla p = -\rho_0(2xy^2\mathbf{i} + 2y^3\mathbf{j})$ , or:  $\frac{\partial p}{\partial x} = -\rho_0 2xy^2$  *Ans.*

**4.28** If  $z$  is "up," what are the conditions on constants  $a$  and  $b$  for which the velocity field  $u = ay$ ,  $v = bx$ ,  $w = 0$  is an exact solution to the continuity and Navier-Stokes equations for incompressible flow?

**Solution:** First examine the continuity equation:

$$\frac{\partial u}{\partial x} + \frac{\partial v}{\partial y} + \frac{\partial w}{\partial z} \stackrel{?}{=} 0 = \frac{\partial}{\partial x}(ay) + \frac{\partial}{\partial y}(bx) + \frac{\partial}{\partial z}(0) = 0 + 0 + 0 \quad \text{for all } a \text{ and } b$$

Given  $g_x = g_y = 0$  and  $w = 0$ , we need only examine  $x$ - and  $y$ -momentum:

$$\rho \left( u \frac{\partial u}{\partial x} + v \frac{\partial u}{\partial y} \right) = -\frac{\partial p}{\partial x} + \mu \left( \frac{\partial^2 u}{\partial x^2} + \frac{\partial^2 u}{\partial y^2} \right) = \rho [(ay)(0) + (bx)(a)] = -\frac{\partial p}{\partial x} + \mu(0 + 0)$$

$$\rho \left( u \frac{\partial v}{\partial x} + v \frac{\partial v}{\partial y} \right) = -\frac{\partial p}{\partial y} + \mu \left( \frac{\partial^2 v}{\partial x^2} + \frac{\partial^2 v}{\partial y^2} \right) = \rho [(ay)(b) + (bx)(0)] = -\frac{\partial p}{\partial y} + \mu(0 + 0)$$