

**8-57** Air is accelerated in a nozzle while losing some heat to the surroundings. The exit temperature of air and the exergy destroyed during the process are to be determined.

**Assumptions** 1 Air is an ideal gas with variable specific heats. 2 The nozzle operates steadily.

**Properties** The gas constant of air is  $R = 0.287 \text{ kJ/kg}\cdot\text{K}$  (Table A-1). The properties of air at the nozzle inlet are (Table A-17)

$$T_1 = 338 \text{ K} \longrightarrow h_1 = 338.40 \text{ kJ/kg}$$

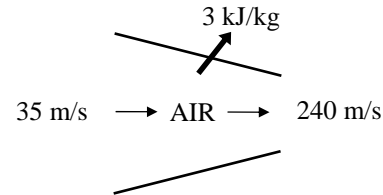
$$s_1^0 = 1.8219 \text{ kJ/kg}\cdot\text{K}$$

**Analysis** (a) We take the nozzle as the system, which is a control volume. The energy balance for this steady-flow system can be expressed in the rate form as

$$\underbrace{\dot{E}_{\text{in}} - \dot{E}_{\text{out}}}_{\text{Rate of net energy transfer by heat, work, and mass}} = \underbrace{\Delta \dot{E}_{\text{system}}^{\neq 0}}_{\text{Rate of change in internal, kinetic, potential, etc. energies}} \stackrel{\neq 0 \text{ (steady)}}{=} 0$$

$$\dot{E}_{\text{in}} = \dot{E}_{\text{out}}$$

$$\dot{m}(h_1 + V_1^2/2) = \dot{m}(h_2 + V_2^2/2) + \dot{Q}_{\text{out}}$$



or

$$0 = q_{\text{out}} + h_2 - h_1 + \frac{V_2^2 - V_1^2}{2}$$

Therefore,

$$h_2 = h_1 - q_{\text{out}} - \frac{V_2^2 - V_1^2}{2} = 338.40 - 3 - \frac{(240 \text{ m/s})^2 - (35 \text{ m/s})^2}{2} \left( \frac{1 \text{ kJ/kg}}{1000 \text{ m}^2/\text{s}^2} \right) = 307.21 \text{ kJ/kg}$$

At this  $h_2$  value we read, from Table A-17,  $T_2 = 307.0 \text{ K} = \mathbf{34.0^\circ\text{C}}$  and  $s_2^0 = 1.7251 \text{ kJ/kg}\cdot\text{K}$

(b) The exergy destroyed during a process can be determined from an exergy balance or directly from its definition

$X_{\text{destroyed}} = T_0 \dot{S}_{\text{gen}}$  where the entropy generation  $\dot{S}_{\text{gen}}$  is determined from an entropy balance on an *extended system* that includes the device and its immediate surroundings so that the boundary temperature of the extended system is  $T_{\text{surr}}$  at all times. It gives

$$\underbrace{\dot{S}_{\text{in}} - \dot{S}_{\text{out}}}_{\text{Rate of net entropy transfer by heat and mass}} + \underbrace{\dot{S}_{\text{gen}}}_{\text{Rate of entropy generation}} = \underbrace{\Delta \dot{S}_{\text{system}}^{\neq 0}}_{\text{Rate of change of entropy}} = 0$$

$$ms_1 - ms_2 - \frac{\dot{Q}_{\text{out}}}{T_{\text{b,surr}}} + \dot{S}_{\text{gen}} = 0$$

$$\dot{S}_{\text{gen}} = \dot{m}(s_2 - s_1) + \frac{\dot{Q}_{\text{out}}}{T_{\text{surr}}}$$

where

$$\Delta s_{\text{air}} = s_2^0 - s_1^0 - R \ln \frac{P_2}{P_1} = (1.7251 - 1.8219) \text{ kJ/kg}\cdot\text{K} - (0.287 \text{ kJ/kg}\cdot\text{K}) \ln \frac{95 \text{ kPa}}{200 \text{ kPa}} = 0.1169 \text{ kJ/kg}\cdot\text{K}$$

Substituting, the entropy generation and exergy destruction per unit mass of air are determined to be

$$x_{\text{destroyed}} = T_0 s_{\text{gen}} = T_{\text{surr}} s_{\text{gen}}$$

$$= T_0 \left( s_2 - s_1 + \frac{q_{\text{surr}}}{T_{\text{surr}}} \right) = (290 \text{ K}) \left( 0.1169 \text{ kJ/kg}\cdot\text{K} + \frac{3 \text{ kJ/kg}}{290 \text{ K}} \right) = \mathbf{36.9 \text{ kJ/kg}}$$

**Alternative solution** The exergy destroyed during a process can be determined from an exergy balance applied on the *extended system* that includes the device and its immediate surroundings so that the boundary temperature of the extended system is environment temperature  $T_0$  (or  $T_{\text{surr}}$ ) at all times. Noting that exergy transfer with heat is zero when the temperature at the point of transfer is the environment temperature, the exergy balance for this steady-flow system can be expressed as

$$\begin{aligned}
 \underbrace{\dot{X}_{\text{in}} - \dot{X}_{\text{out}}}_{\text{Rate of net exergy transfer by heat, work, and mass}} - \underbrace{\dot{X}_{\text{destroyed}}}_{\text{Rate of exergy destruction}} &= \underbrace{\Delta \dot{X}_{\text{system}}^{\phi_0} (\text{steady})}_{\text{Rate of change of exergy}} = 0 \rightarrow \dot{X}_{\text{destroyed}} = \dot{X}_{\text{in}} - \dot{X}_{\text{out}} = \dot{m}\psi_1 - \dot{m}\psi_2 = \dot{m}(\psi_1 - \psi_2) \\
 &= \dot{m}[(h_1 - h_2) - T_0(s_1 - s_2) - \Delta ke - \Delta pe^{\phi_0}] = \dot{m}[T_0(s_2 - s_1) - (h_2 - h_1 + \Delta ke)] \\
 &= \dot{m}[T_0(s_2 - s_1) + q_{\text{out}}] \quad \text{since, from energy balance, } -q_{\text{out}} = h_2 - h_1 + \Delta ke \\
 &= T_0 \left( \dot{m}(s_2 - s_1) + \frac{\dot{Q}_{\text{out}}}{T_0} \right) = T_0 \dot{S}_{\text{gen}}
 \end{aligned}$$

Therefore, the two approaches for the determination of exergy destruction are identical.