

**5-49** Steam expands in a turbine. The change in kinetic energy, the power output, and the turbine inlet area are to be determined.

**Assumptions 1** This is a steady-flow process since there is no change with time. **2** Potential energy changes are negligible. **3** The device is adiabatic and thus heat transfer is negligible.

**Properties** From the steam tables (Tables A-4 through 6)

$$\left. \begin{array}{l} P_1 = 6 \text{ MPa} \\ T_1 = 400^\circ\text{C} \end{array} \right\} \begin{array}{l} v_1 = 0.047420 \text{ m}^3/\text{kg} \\ h_1 = 3178.3 \text{ kJ/kg} \end{array}$$

and

$$\left. \begin{array}{l} P_2 = 40 \text{ kPa} \\ x_2 = 0.92 \end{array} \right\} h_2 = h_f + x_2 h_{fg} = 317.62 + 0.92 \times 2392.1 = 2318.5 \text{ kJ/kg}$$

**Analysis** (a) The change in kinetic energy is determined from

$$\Delta ke = \frac{V_2^2 - V_1^2}{2} = \frac{(50 \text{ m/s})^2 - (80 \text{ m/s})^2}{2} \left( \frac{1 \text{ kJ/kg}}{1000 \text{ m}^2/\text{s}^2} \right) = \mathbf{-1.95 \text{ kJ/kg}}$$

(b) There is only one inlet and one exit, and thus  $\dot{m}_1 = \dot{m}_2 = \dot{m}$ . We take the turbine as the system, which is a control volume since mass crosses the boundary. The energy balance for this steady-flow system can be expressed in the rate form as

$$\underbrace{\dot{E}_{\text{in}} - \dot{E}_{\text{out}}}_{\text{Rate of net energy transfer by heat, work, and mass}} = \underbrace{\Delta \dot{E}_{\text{system}}}_{\text{Rate of change in internal, kinetic, potential, etc. energies}} \stackrel{\text{no (steady)}}{=} 0$$

$$\dot{E}_{\text{in}} = \dot{E}_{\text{out}}$$

$$\dot{m}(h_1 + V_1^2/2) = \dot{W}_{\text{out}} + \dot{m}(h_2 + V_2^2/2) \quad (\text{since } \dot{Q} \cong \Delta pe \cong 0)$$

$$\dot{W}_{\text{out}} = -\dot{m} \left( h_2 - h_1 + \frac{V_2^2 - V_1^2}{2} \right)$$

Then the power output of the turbine is determined by substitution to be

$$\dot{W}_{\text{out}} = -(20 \text{ kg/s})(2318.5 - 3178.3 - 1.95) \text{ kJ/kg} = 14,590 \text{ kW} = \mathbf{14.6 \text{ MW}}$$

(c) The inlet area of the turbine is determined from the mass flow rate relation,

$$\dot{m} = \frac{1}{v_1} A_1 V_1 \longrightarrow A_1 = \frac{\dot{m} v_1}{V_1} = \frac{(20 \text{ kg/s})(0.047420 \text{ m}^3/\text{kg})}{80 \text{ m/s}} = \mathbf{0.0119 \text{ m}^2}$$

