

# Effective thermal conductivity of adsorption packed beds

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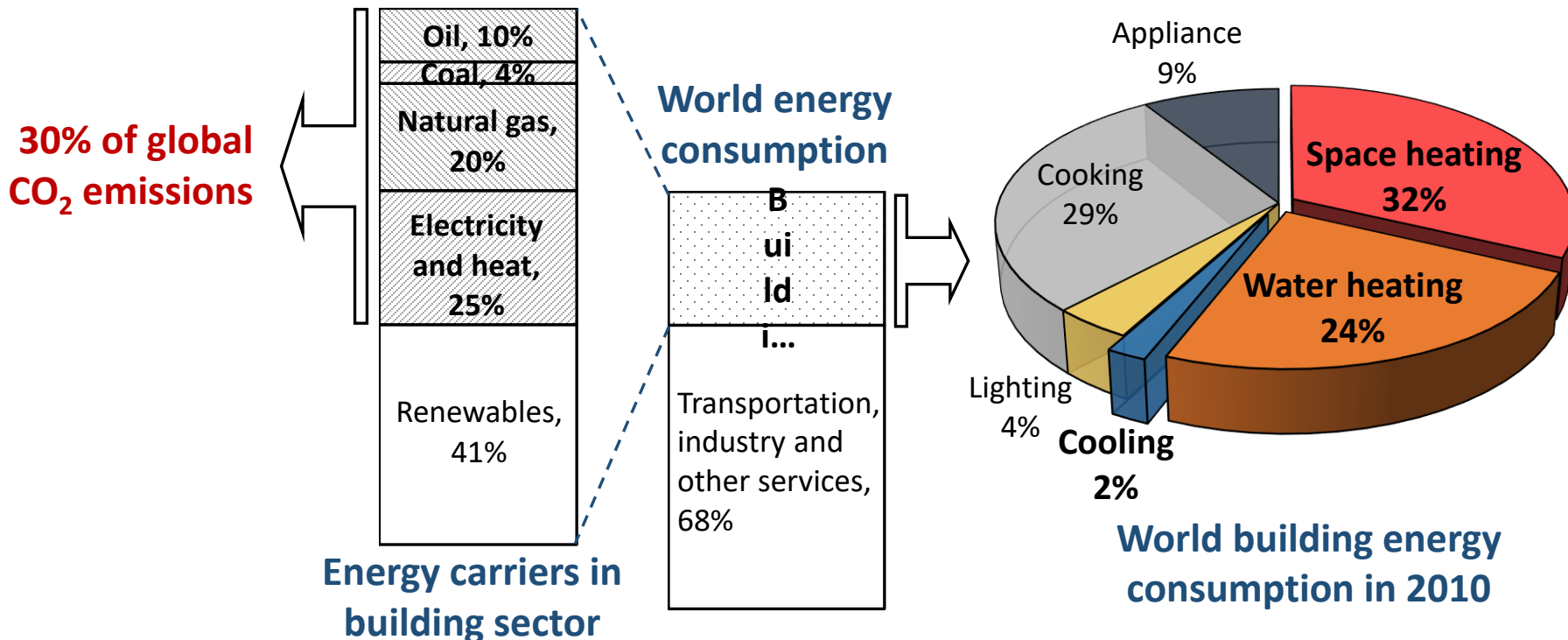
Greater Vancouver region, Canada

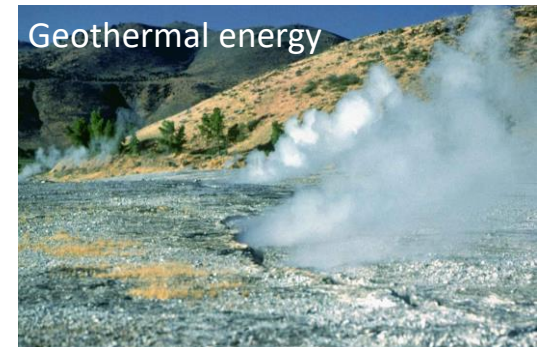


Simon Fraser University, Surrey Campus

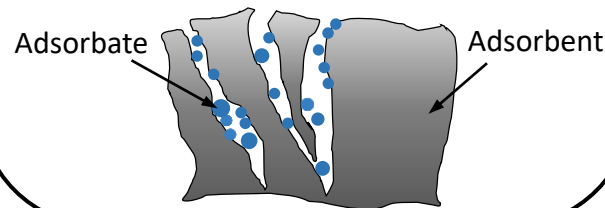
- Transport phenomena in fuel cell catalyst layers
- Passive cooling systems for power electronics
- Thermal management of batteries
- Efficient HVAC-R
- Atmospheric water generation
- **Sorption chillers, thermal storage, dehumidification**

- **32% (115 EJ) of global energy demand** is used in **building sector**.
- Energy consumption for **heating and cooling** in building sector:  
32% space heating, 24% water heating and 2% cooling
- **60%** of the energy consumption in buildings is **non-renewables** and **CO<sub>2</sub> emission producers**.
- About **one-third of global CO<sub>2</sub> emissions** is produced by **buildings sector**.





## Adsorption Storage Systems

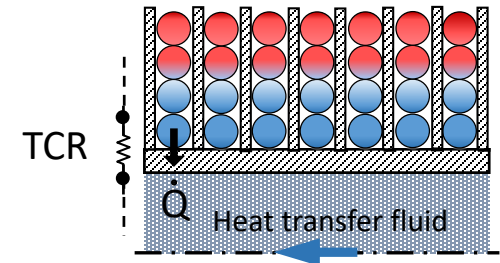


Useful Work

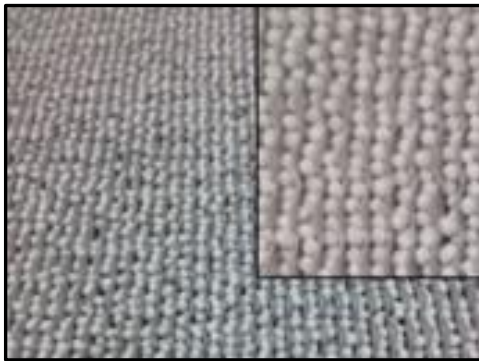
- ✓ Efficient use of energy (use of low-grade heat, below 100 °C)
- ✓ Effective climate protection through reduced CO<sub>2</sub> emissions

### Limiting factors against adoption of adsorption systems:

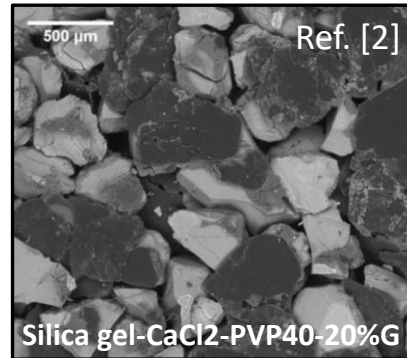
- ❑ Low mass transfer
- ❑ Low heat transfer
  - Low thermal conductivity of adsorbent materials ( $0.1\text{-}0.4\text{ W}\cdot\text{m}^{-1}\cdot\text{K}^{-1}$ )
  - High thermal contact resistance (TCR) between the adsorbent material and the heat exchanger (HEX) metal surface.



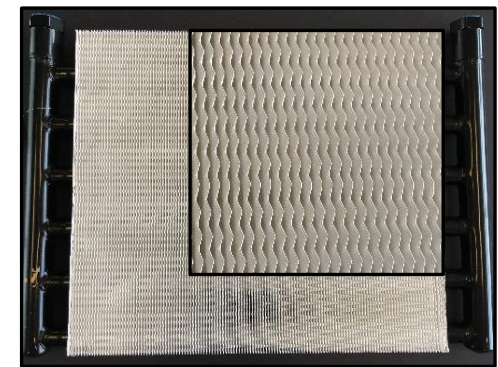
### Contact types of adsorbent materials with HEX media:



**Granular (loose grains)**



**Consolidated**



**Coated HEX**

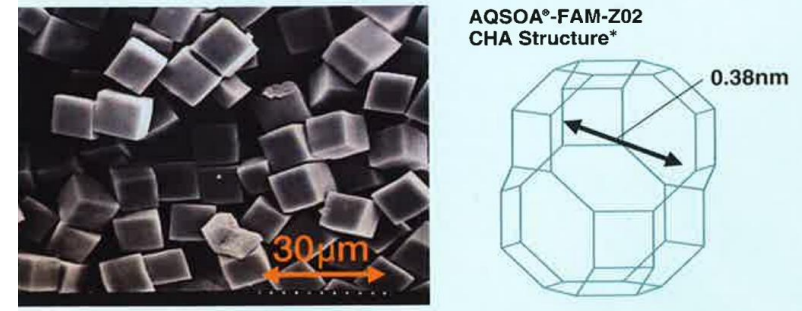
- ✓ Simple and cheap
- ✓ Higher volumetric power  $\left(\frac{Q_{delivered}}{t \cdot V}\right)$  [3] and higher energy storage density  $\left(\frac{Q_{delivered}}{V}\right)$

[2] K. Fayazmanesh, C. McCague, and M. Bahrami, *Appl. Therm. Eng.* (123) 753–760, 2017.

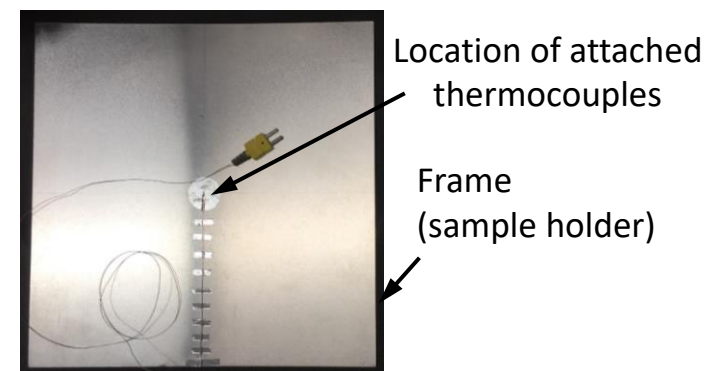
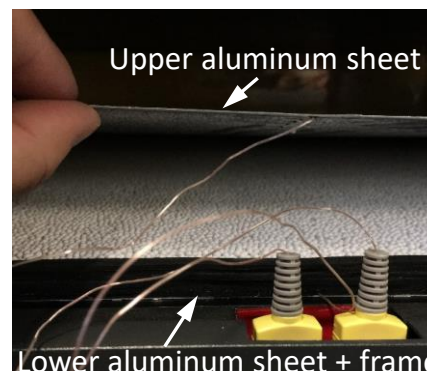
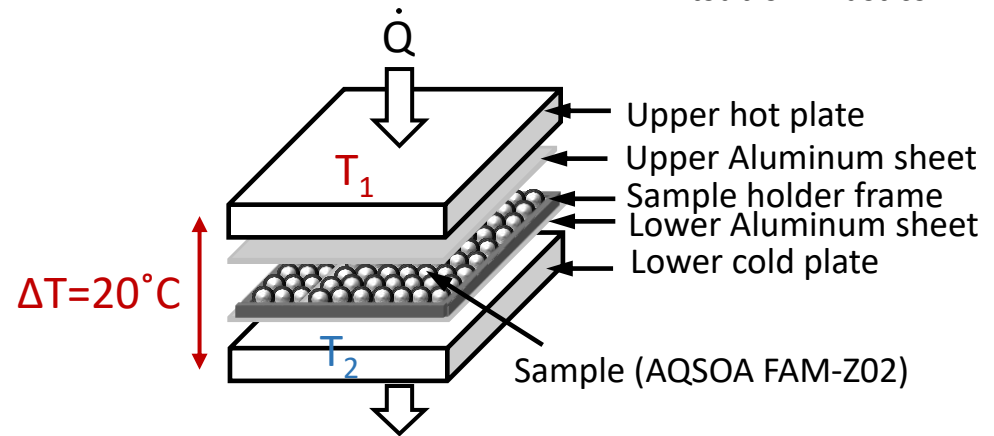
[3] A. Freni, L. Bonaccorsi, L. Calabrese, A. Capri, A. Frazzica, and A. Sapienza, *Appl. Therm. Eng.* (82) 1–7, 2015.

- **Measure** the **effective thermal conductivity** of a **AQSOA FAM-Z02** packed bed
- **Model** the **effective bed thermal conductivity** of packed bed adsorber
- Investigate the effects of **key parameters** on the thermal conductivity, including **temperature, uptake** and **numbers of adsorbent layers**
- Study the importance of **thermal contact resistance (TCR)** between the adsorbent medium and heat exchanger metal surface

- **Adsorbent material: AQSOA FAM-Z02**  
(2 mm diameter beads, Mitsubishi Plastics)
- **Apparatus:**
  - Heat flow meter (HFM 436/3/1E, Netzsch)
  - Suitable for low conductivity materials  
**(0.002 to 2.0 W·m<sup>-1</sup>·K<sup>-1</sup>)**
  - Accuracy of **± 1-3%**
  - ASTM C518 standard

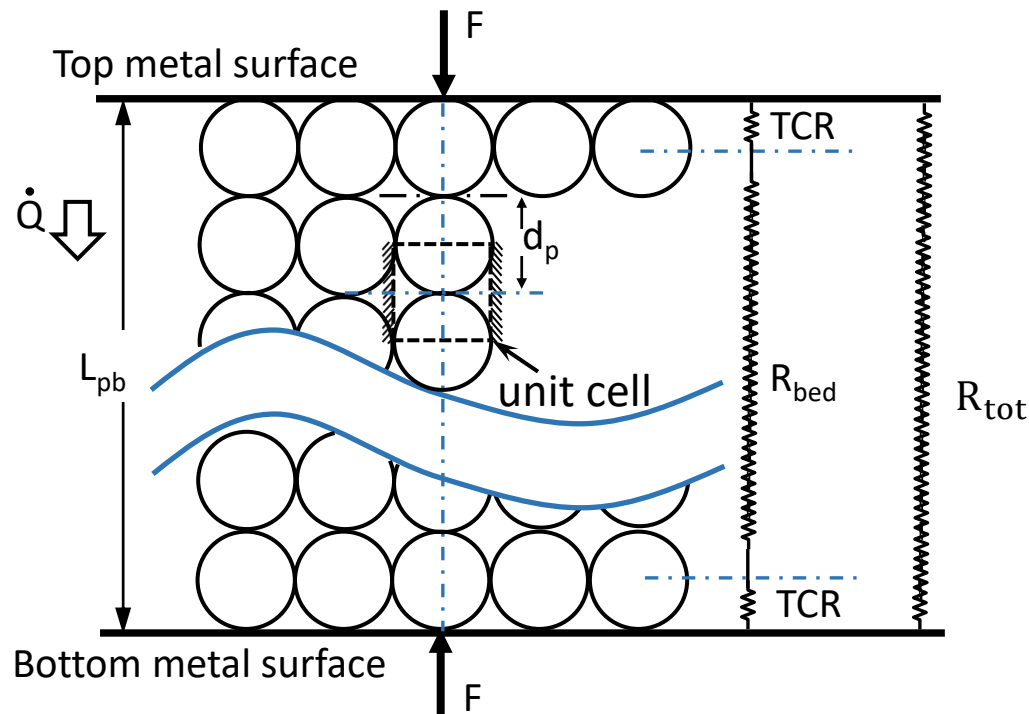
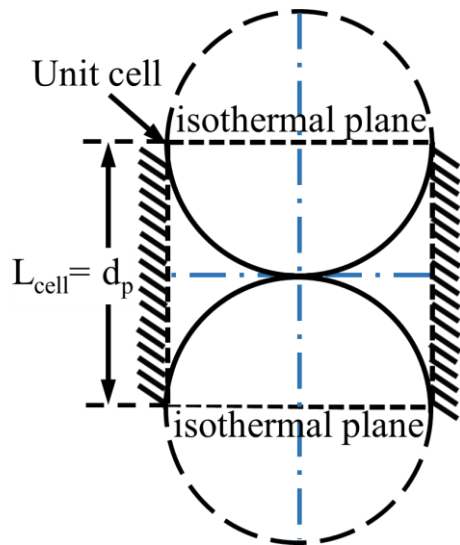


Mitsubishi Plastics



Predicting effective thermal conductivity of an adsorption packed bed as a function of:

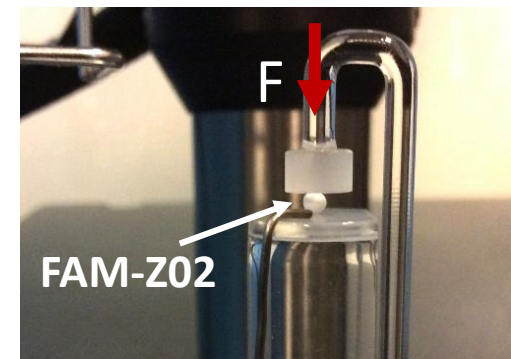
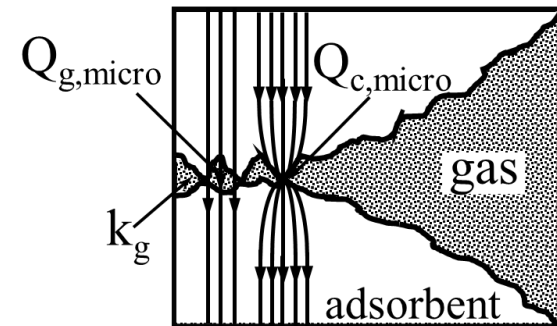
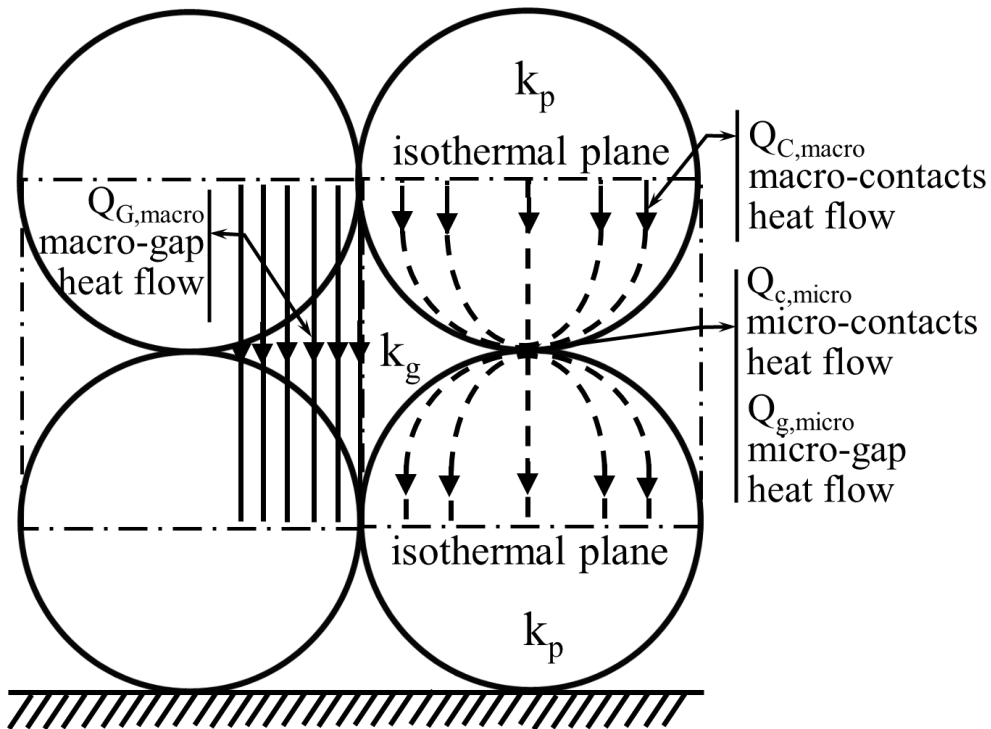
- Temperature
- Contact pressure
- Particle size
- Gas pressure
- **Packed bed arrangement**
- **Water uptake**



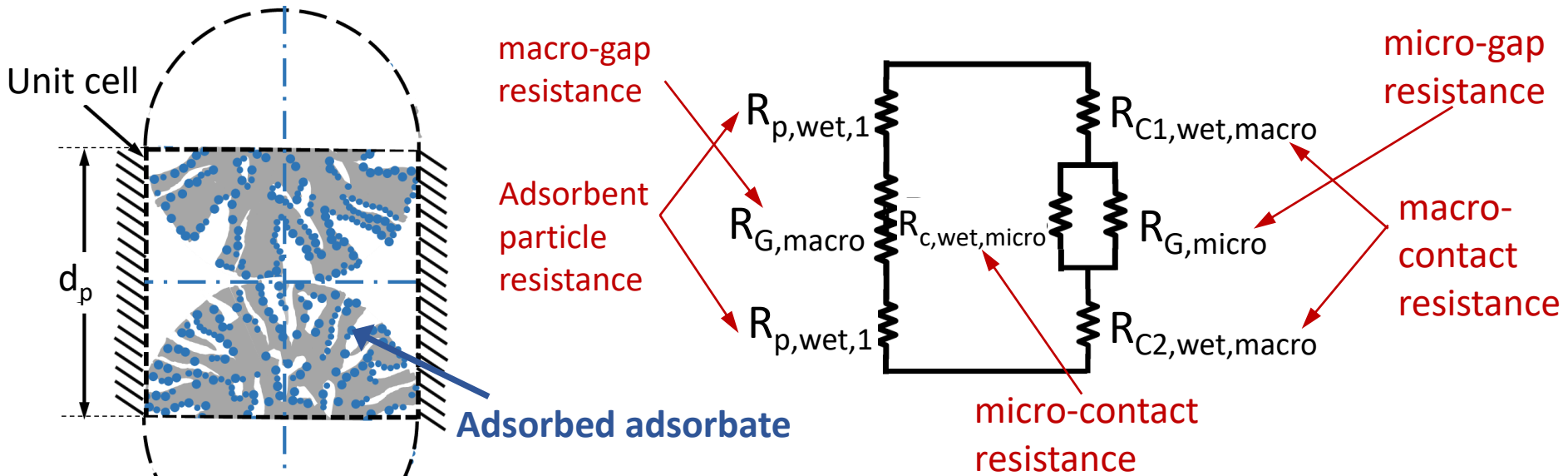


Two main paths for heat transfer in the packed bed:

- Conduction through solid adsorbent, i.e., a) macro-contact, b) micro-contact heat transfer
- Conduction through interstitial gas, i.e., a) macro-gap, b) micro-gap gas heat transfer



Thermomechanical analyzer  
(TMA Q400EM, TA Instruments)



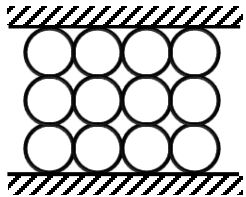
$$R_{unit\ cell} = \left[ \frac{1}{(1/R_{c,wet,micro} + 1/R_{g,micro})^{-1} + R_{C,wet,macro}} + \frac{1}{R_{G,macro}} \right]^{-1}$$

- Bruggeman's method based on the assumption of **effective homogeneous medium**:

$$\sum_{i=1}^3 \beta_i \frac{k_i - k_{p,wet}}{k_i + 2k_{p,wet}} = 0, \quad \text{where } \sum_{i=1}^3 \beta_i = 1 \quad \left\{ \begin{array}{l} \beta_s = 1 - \varepsilon \quad \text{Solid particle} \\ \beta_w = \omega \frac{\rho_s}{\rho_w} (1 - \varepsilon) \quad \text{Adsorbed water} \\ \beta_g = \varepsilon - \omega \frac{\rho_s}{\rho_w} (1 - \varepsilon) \quad \text{Filling gas} \end{array} \right.$$

- The model is applicable to the following packed bed arrangements:

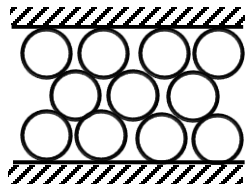
I. Simple cubic (SC)



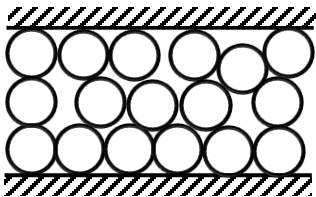
Packing	Solid fraction, $\psi$	Bed length, $L_{bed}$	Cell length, $L_{cell}$
SC	0.524	$n \times d_p$	$d_p$
FCC	0.740	$((n - 1)\sqrt{2}/2 + 1)d_p$	$\sqrt{2}d_p/2$

\* n: number of layers

II. Face-centered cubic (FCC)



III. Randomly packed bed adsorber



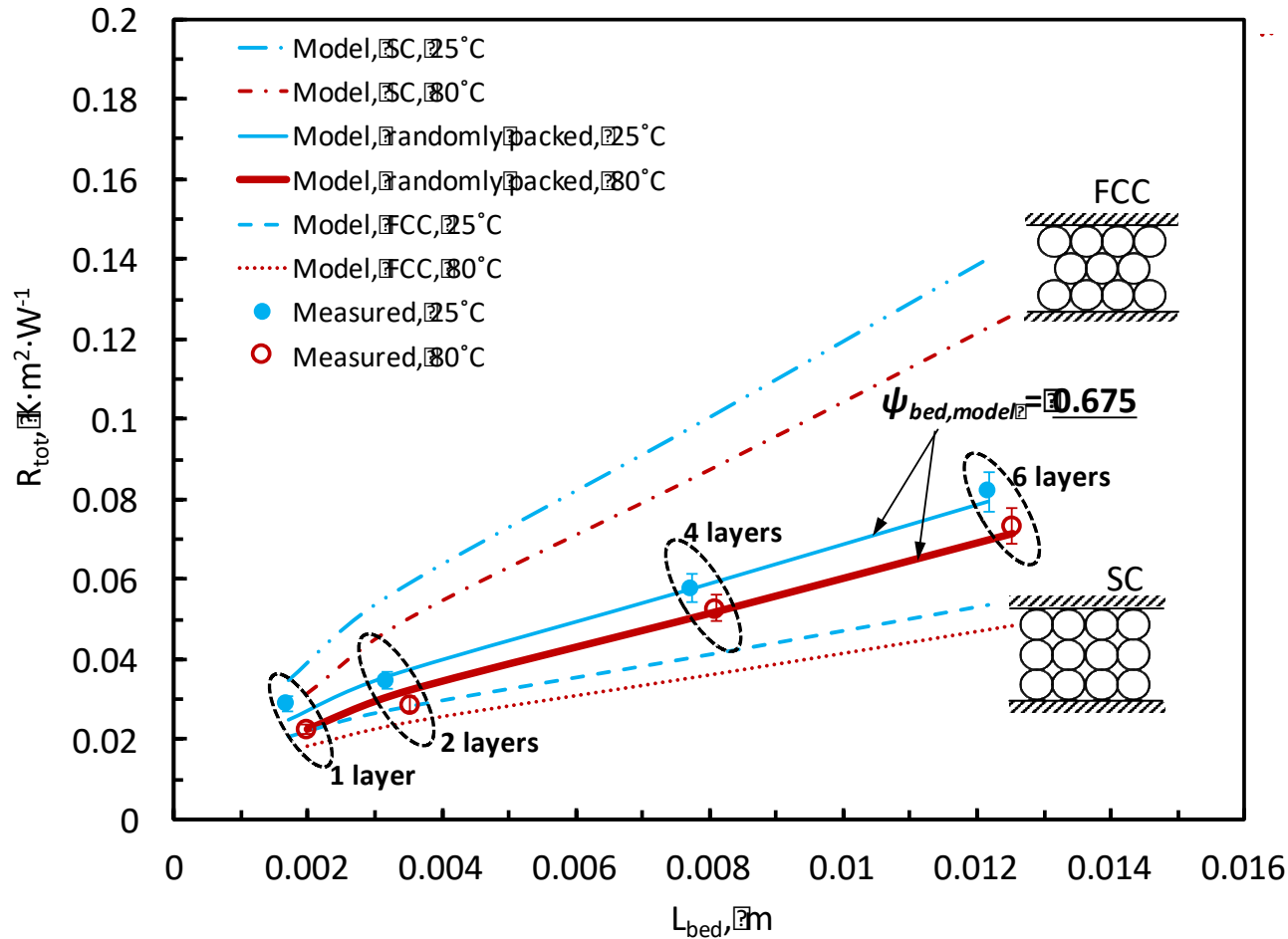
$$\psi_{FCC} < \psi_{bed} < \psi_{SC}$$

$$k_{FCC} < k_{bed} < k_{SC}$$

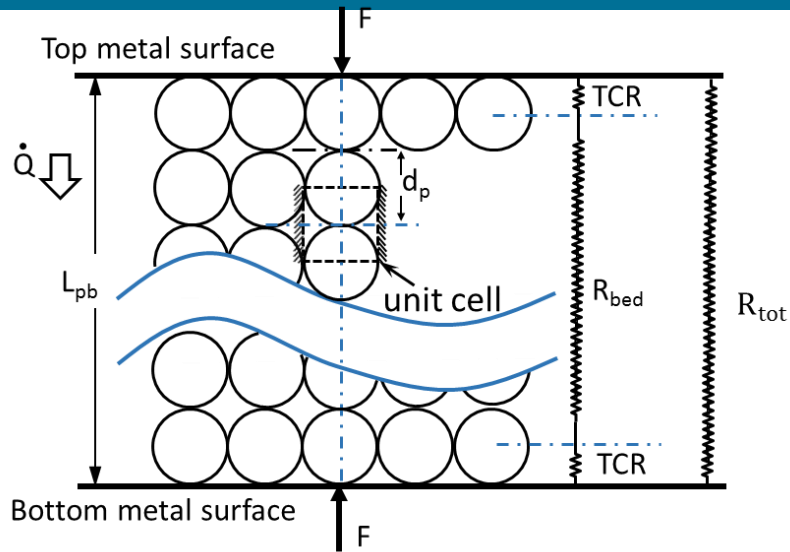
linear asymptotic solution:

$$\frac{\psi_{bed} - \psi_{SC}}{\psi_{FCC} - \psi_{SC}} = \frac{k_{bed} - k_{SC}}{k_{FCC} - k_{SC}}$$

# Asymptotic solution for intermediate solid fraction (randomly packed)



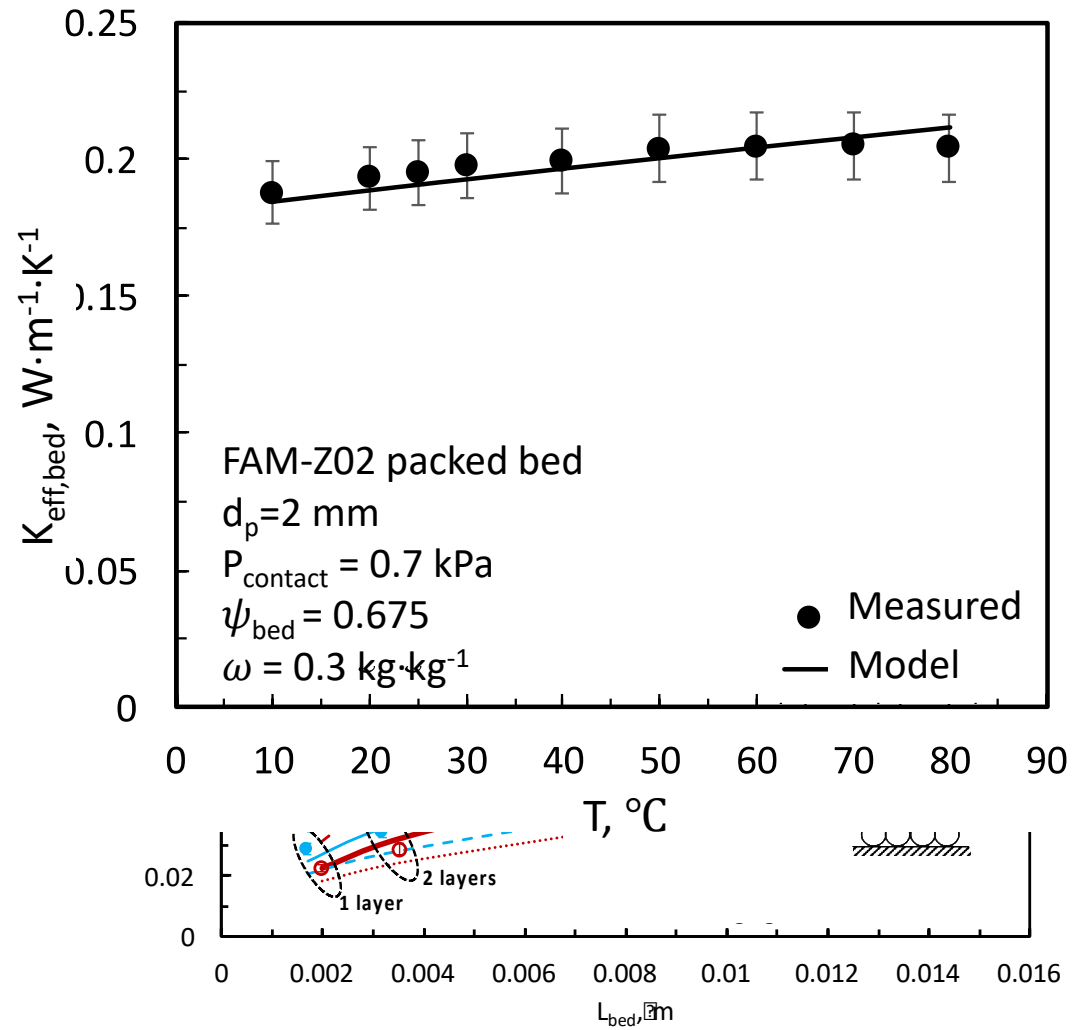
- **SC is the lower limit of thermal resistance** of the packed bed.  
**FCC is the upper limit of thermal resistance** of the packed bed.
- **Solid fraction of the bed is 0.675.**
- **Thermal resistance at 25°C is higher than 80°C**, due to lower thermal conductivity of adsorbent and gas at lower temperatures.



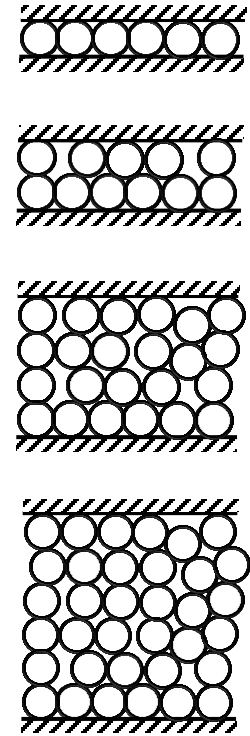
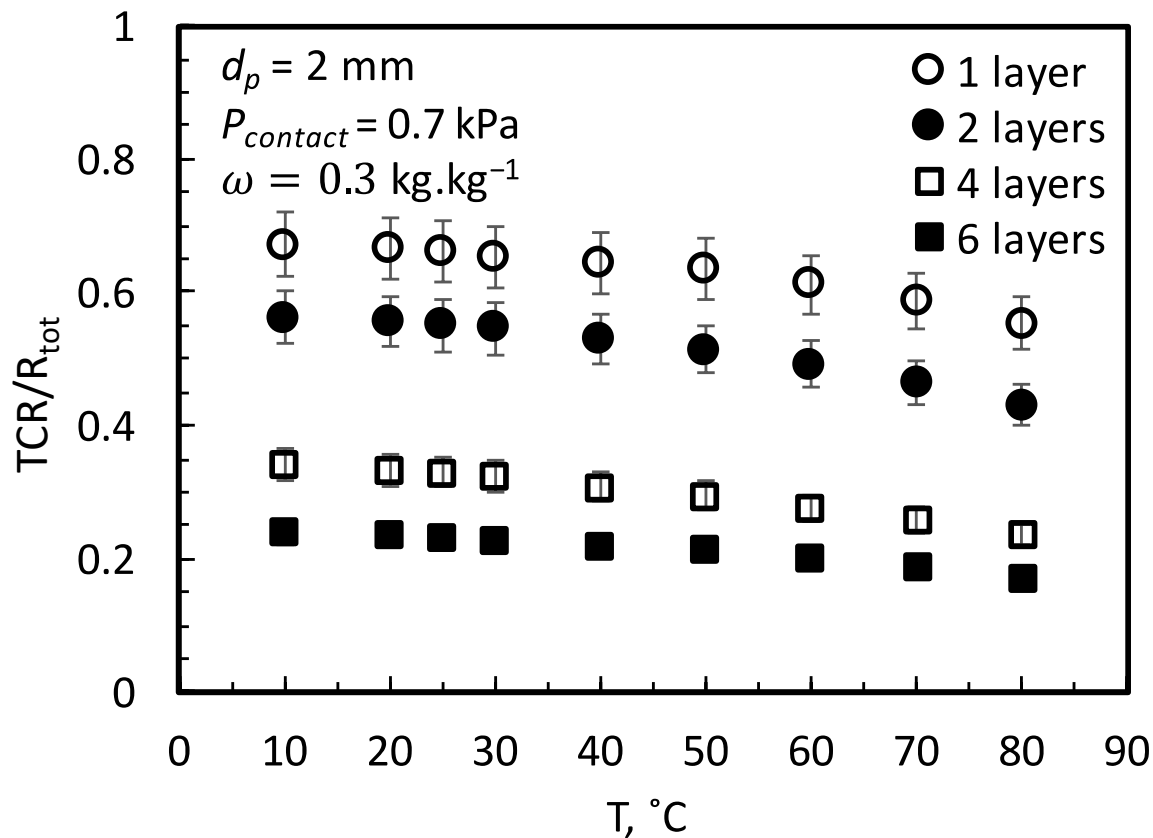
$$R_{tot,1} = \frac{L_{bed,1}}{k_{eff,bed} A} + TCR$$

$$R_{tot,2} = \frac{L_{bed,2}}{k_{eff,bed} A} + TCR$$

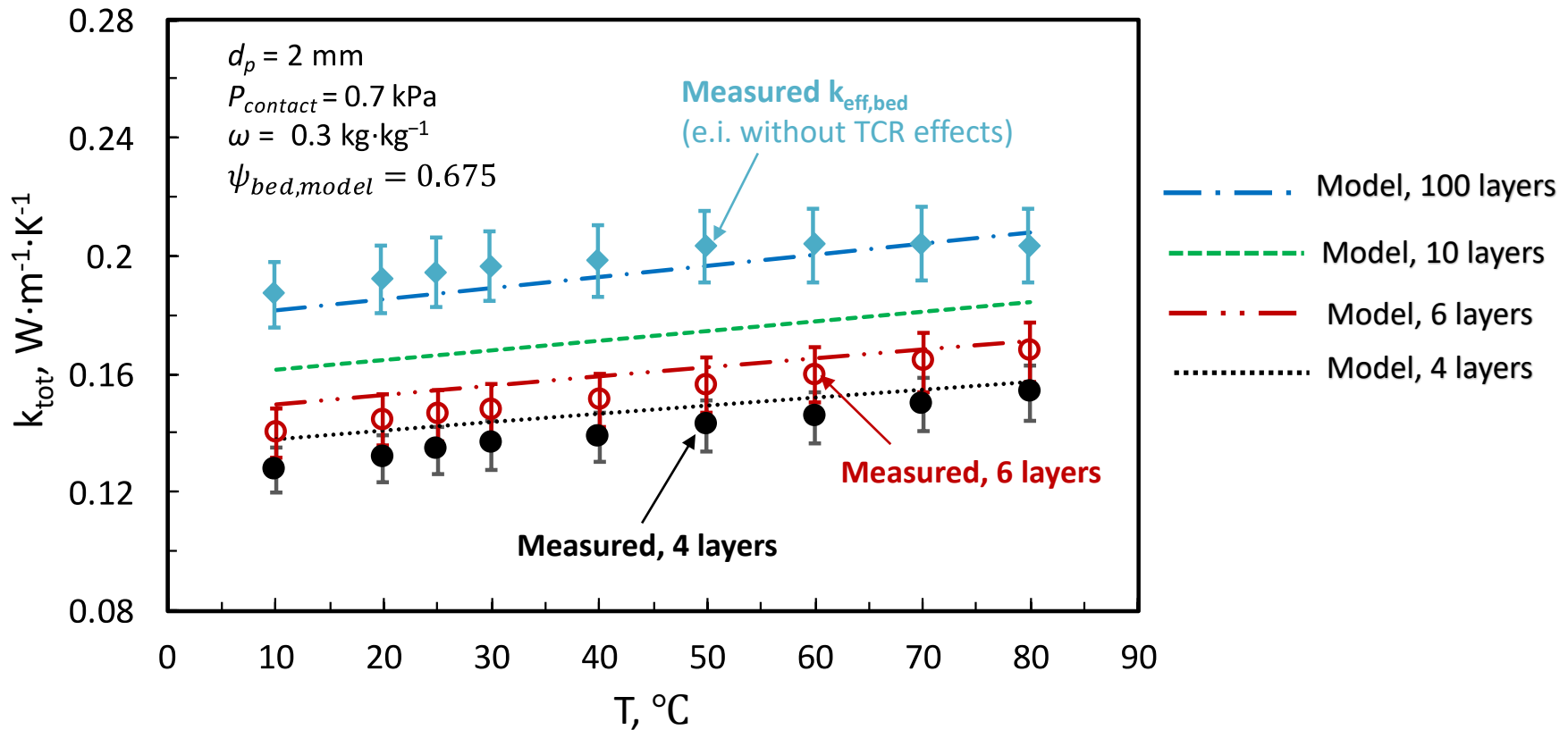
$$k_{eff,bed} = \frac{A(R_{tot,2} - R_{tot,1})}{(L_2 - L_1)}$$



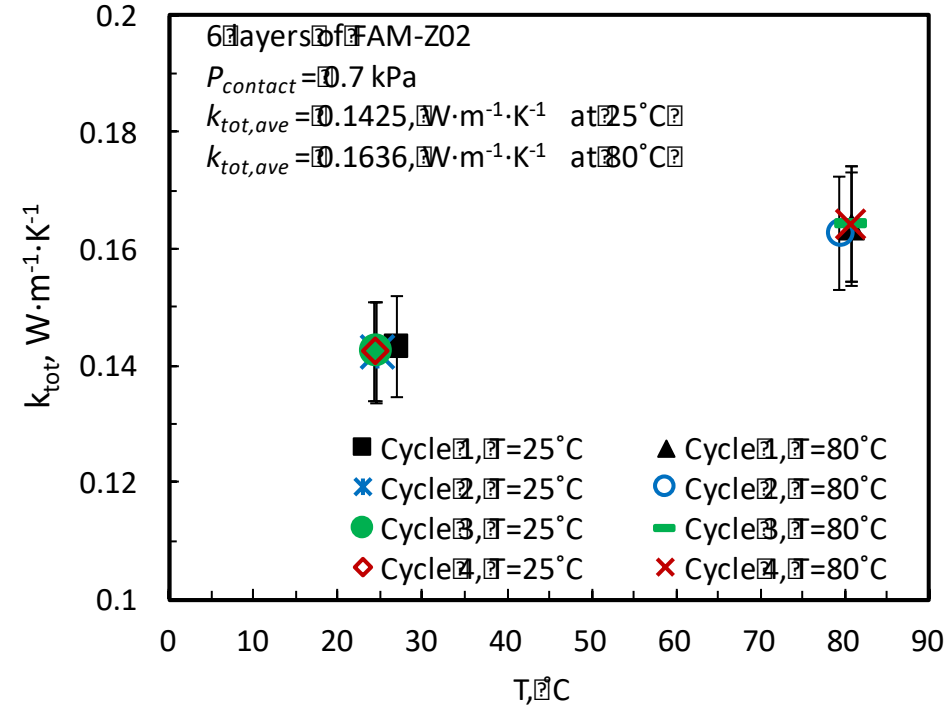
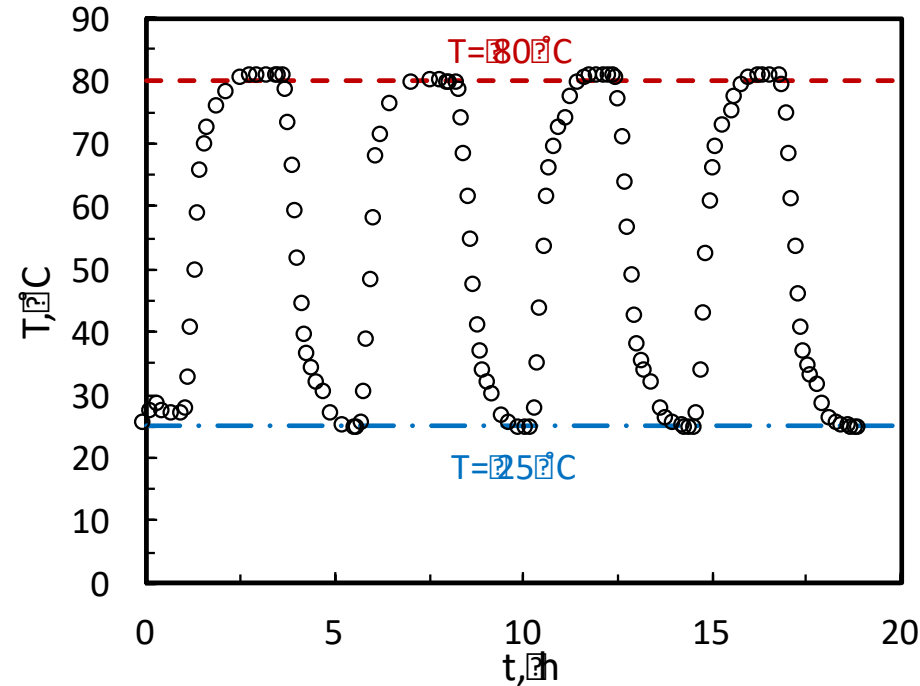
- $1/(k_{eff,bed} \cdot A)$  is the **slope** of the plot of  $R_{tot}$  versus  $L_{bed}$ .
- **TCR** is the **intercept** of the plot of  $R_{tot}$  versus  $L_{bed}$ .



- $\text{TCR}/R_{\text{tot}}$  is greater for **lower number of adsorbent layers** and **lower temperatures**.
- At **higher temperatures**, TCR decreases due to
  - i) **higher thermal conductivity** of adsorbent and filling gas;
  - ii) **better contact** at the interface due to **the thermal expansion**

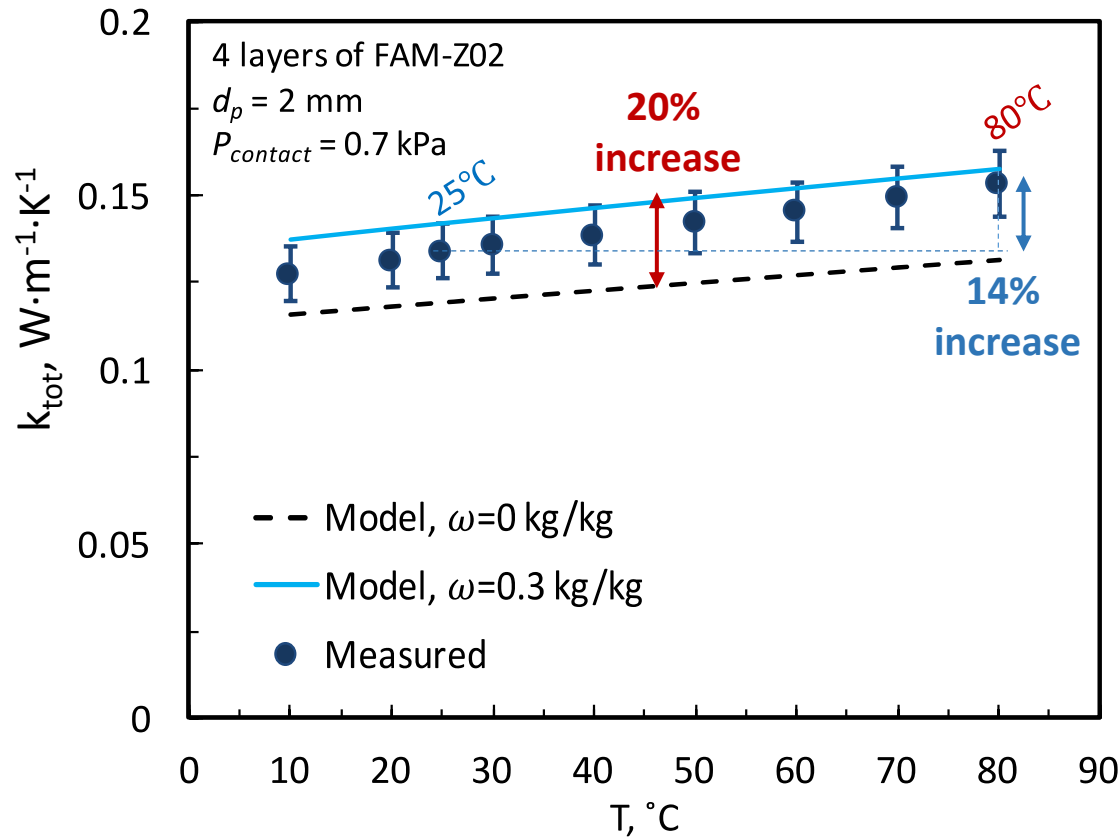


- Total bed thermal conductivity **increases** as **temperature increases** due to the **lower TCR**.
- For **multi-layer adsorbent**, the thermal behavior approaches the **“effective” or continuum medium** thermal conductivity.



- Negligible hysteresis in thermal conductivity for the cyclic temperature changes between  $25^{\circ}\text{C}$  (adsorption) and  $80^{\circ}\text{C}$  (desorption).
- Maximum relative difference is 1%, for both adsorption and desorption, for cyclic temperature changes between 25 to  $80^{\circ}\text{C}$ .





- **20% increase** in the total thermal conductivity by increase in the **uptake from 0 to 0.3 kg·kg<sup>-1</sup>**.
- **14% increase** in the total thermal conductivity by increase in the **temperature from 25°C to 80°C**.

- Effective thermal conductivity of FAM-Z02 packed bed adsorber is measured.
- TCR is deconvoluted from the total bed thermal resistance.
- Importance of thermal contact resistance to the total resistance is calculated.
- Thermal conductivity and TCR of packed bed adsorber are modeled as a function of temperature, bed arrangement, gas pressure and number of adsorbent layers.
- Young's modulus of FAM-Z02 is measured.

# Thank you for your attention!

