## Simple Heating and Cooling

**14-67C** Relative humidity decreases during a simple heating process and increases during a simple cooling process. Specific humidity, on the other hand, remains constant in both cases.

**14-68C** Because a horizontal line on the psychrometric chart represents a  $\omega$  = constant process, and the moisture content  $\omega$  of air remains constant during these processes.

**14-69** Humid air at a specified state is cooled at constant pressure to the dew-point temperature. The cooling required for this process is to be determined.

Assumptions 1 This is a steady-flow process and thus the mass flow rate of dry air remains constant during the entire process  $(\dot{m}_{a1} = \dot{m}_{a2} = \dot{m}_a)$ . 2 Dry air and water vapor are ideal gases. 3 The kinetic and potential energy changes are negligible.

*Analysis* The amount of moisture in the air remains constant ( $\omega_1 = \omega_2$ ) as it flows through the cooling section since the process involves no humidification or dehumidification. The inlet and exit states of the air are completely specified, and the total pressure is 1 atm. The properties of the air at the inlet state are determined from the psychrometric chart (Figure A-31) to be

 $h_1 = 71.3 \text{ kJ/kg dry air}$   $\omega_1 = 0.0161 \text{ kg H}_2\text{O/kg dry air} (= \omega_2)$  $T_{dp,1} = 21.4^{\circ}\text{C}$ 

The exit state enthalpy is

$$\left. \begin{array}{l} P = 1 \text{ atm} \\ T_2 = T_{dp,1} = 21.4^{\circ}\text{C} \\ \phi_2 = 1 \end{array} \right\} h_2 = 62.4 \text{ kJ/kg dry air}$$

From the energy balance on air in the cooling section,

$$q_{\text{out}} = h_1 - h_2 = 71.3 - 62.4 = 8.9 \text{ kJ/kg dry air}$$

