

Lecture 18 · Refrigerators.

- A refrigerator:

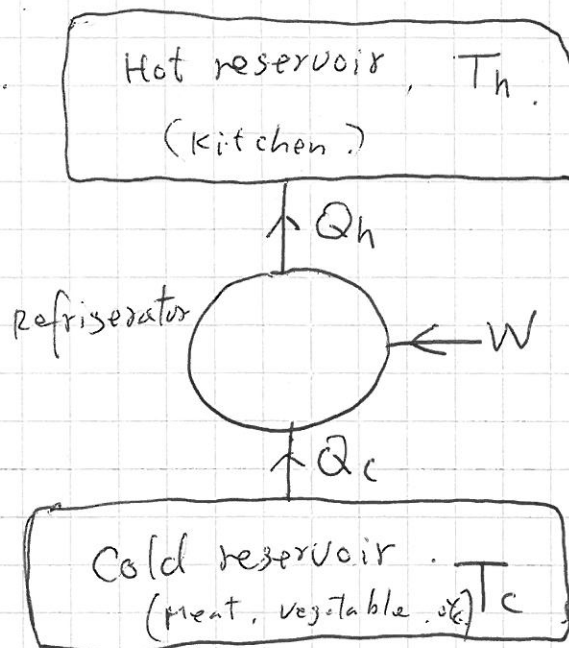
Heat engine operated in reverse.

- Coefficient of performance.

$$COP = \frac{Q_c}{W} = \frac{\text{benefit}}{\text{cost}}.$$

$$(1st\ law): \quad = \frac{Q_c}{Q_h - Q_c}.$$

$$= \frac{1}{Q_h/Q_c - 1}.$$



$$W + Q_c = Q_h. \quad (\Delta U = 0) \quad \uparrow \quad \text{cyclic}.$$

2nd Law: $\Delta S_{\text{Total}} \geq 0.$

$$\frac{Q_h}{T_h} - \frac{Q_c}{T_c} \geq 0$$

$$\left(\Delta S_{\text{refrigerator}} = 0 \right) \quad \because \text{cyclic}.$$

$$\frac{Q_h}{Q_c} \geq \frac{T_h}{T_c}.$$

$$\therefore COP \leq \frac{1}{T_h/T_c - 1} \quad \text{OR} \quad COP \leq \frac{T_c}{T_h - T_c}.$$

$$\begin{cases} < : \text{irreversible} \\ = : \text{reversible} \end{cases}$$

e.g. $T_h = 298 \text{ K}$, $T_c = 255 \text{ K}$.

$$\text{COP} = \frac{255}{298 - 255} = 5.93$$

$\therefore$ COP can be greater than 1 !

$\therefore$ Mechanical energy is more "valuable" than thermal energy.

e.g. Idealized Stirling refrigerator. $\left\{ \begin{array}{l} \text{ideal gas} \\ \text{quasistatic} \\ \text{(reversible)} \\ \text{(no friction)} \end{array} \right\}$

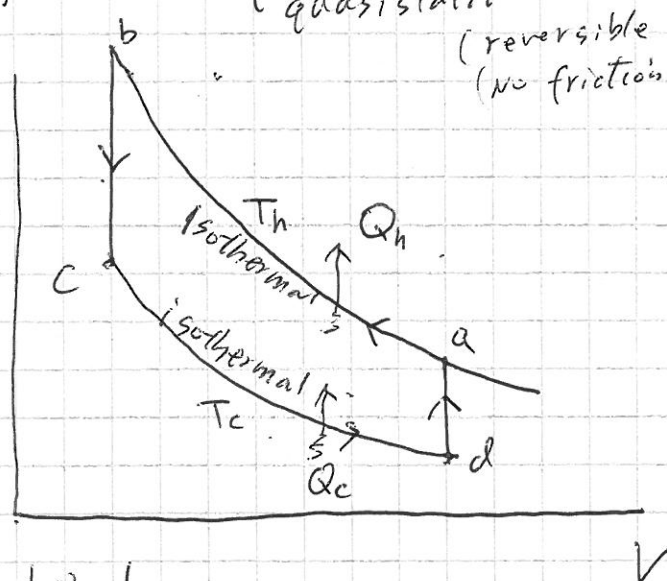
$d \rightarrow a$: No work. ($V = \text{const}$). P
 $\Delta U = C_v (T_h - T_c) = Q_{ad}$

~~absorbed~~
 ↑
 absorbed

$c \rightarrow b$: No work.

$$\Delta U = C_v (T_c - T_h) = Q_{bc}$$

↑
 release $|Q_{bc}|$



~~Net heat~~ $\therefore Q_{ad}$ and Q_{bc} cancel

Net effect of heat: the refrigerator ~~absorbs~~ heat Q_c from the cold reservoir and release heat Q_h to the hot reservoir.

Price ~~we~~ have to pay: we need to do work W on the refrigerator

$$W_{ab} = - \int_a^b p dV$$

$$pV = nRT_h$$

$$p = \frac{nRT_h}{V}$$

$$= -nRT_h \ln \frac{V_b}{V_a}$$

$$W_{cd} = - \int_c^d p dV = -nRT_c \ln \frac{V_d}{V_c} = -nRT_c \ln \frac{V_a}{V_b}$$

$$(V_a = V_d \quad V_c = V_b)$$

$$W = W_{ab} + W_{cd} = -nRT_h \ln \frac{V_b}{V_a} + nRT_c \ln \frac{V_b}{V_a}$$

$$= \left(nR \ln \frac{V_b}{V_a} \right) (T_c - T_h)$$

$$= (T_h - T_c) nR \ln \frac{V_a}{V_b}$$

$$Q_c = -W_{cd} \quad (\because \Delta U_{cd} = 0 \text{ isothermal})$$

$$= nRT_c \ln \frac{V_a}{V_b}$$

$$\text{COP} = \frac{Q_c}{W} = \frac{T_c}{T_h - T_c}$$

• Real Heat engines .

idealized machines — give the limit in efficiency and COP .

(but maybe slow .
or inconvenient .)

• Internal Combustion Engines .

Gasoline engine in your car .

Otto cycle .

1 — injection of air + fuel vapour .

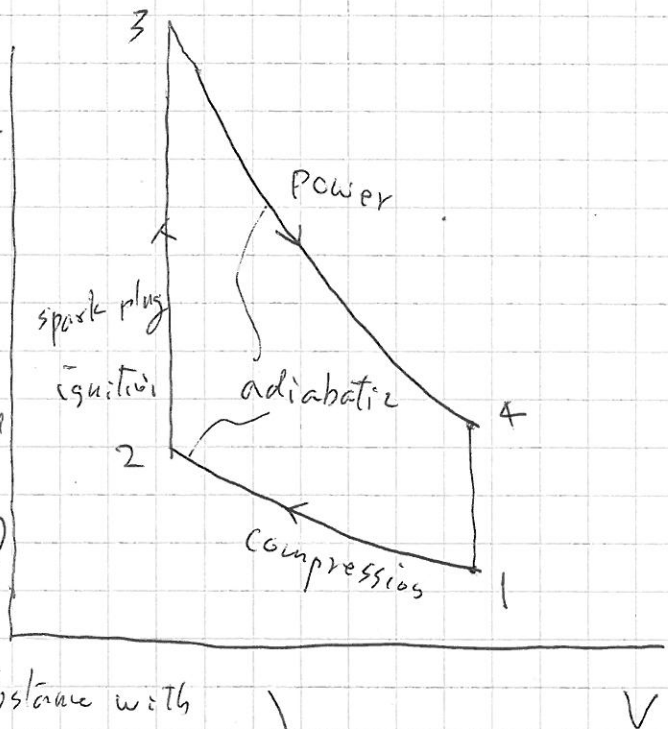
1 → 2 compression (adiabatic)

2 — ignition .

2 → 3 pressure increase which $V = \text{fixed}$ (burning of fuel)

3 → 4 power (adiabatic expansion)

4 → 1 exhaust .



(replace old working substance with
the new one, without doing work)

Efficiency . (assignment 6. # 4.18)

$$e = 1 - \left(\frac{V_2}{V_1} \right)^{\gamma-1} . \quad \frac{V_1}{V_2} = \text{compression ratio}$$

Note: $\frac{V_1}{V_2} \sim$ depends on T_1 and T_2 . (T_3, T_4)

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e.g. $\gamma = 7/5$. $\frac{V_1}{V_2} = 8$. (Typical)

$$e = 1 - \left(\frac{1}{8}\right)^{2/5} = 0.56$$

The higher the $\frac{V_1}{V_2}$, the higher the e .

but there is a limit: if $\frac{V_1}{V_2}$ is too high,

the fuel would burn before you turn on the spark plug.

Solution: Diesel engine.

Typically $\frac{V_1}{V_2} \approx 20$. (limited by material of the piston, cylinder.

$$T_2 = 820 \text{ K}$$

